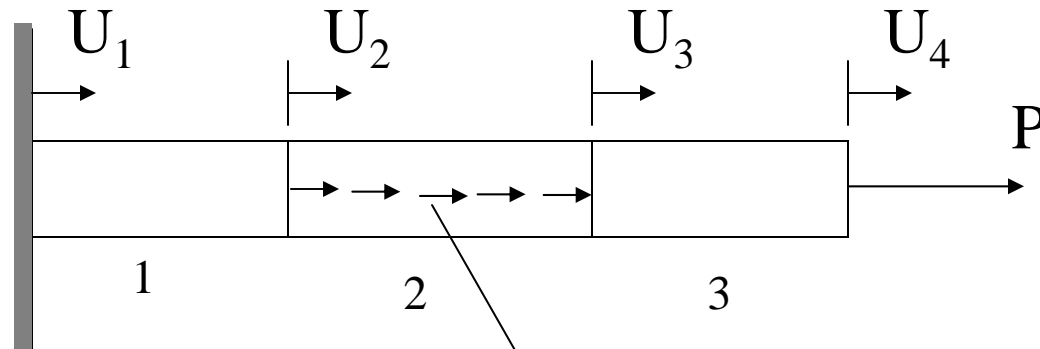
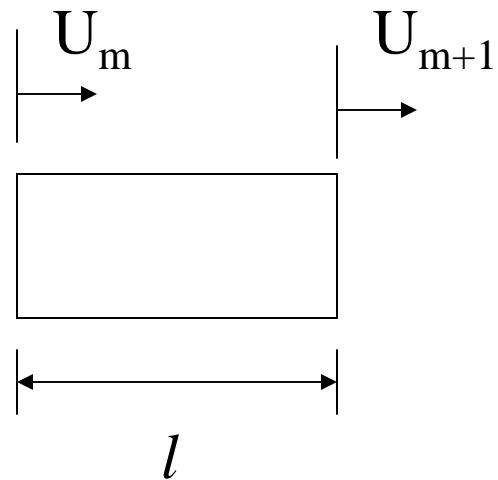


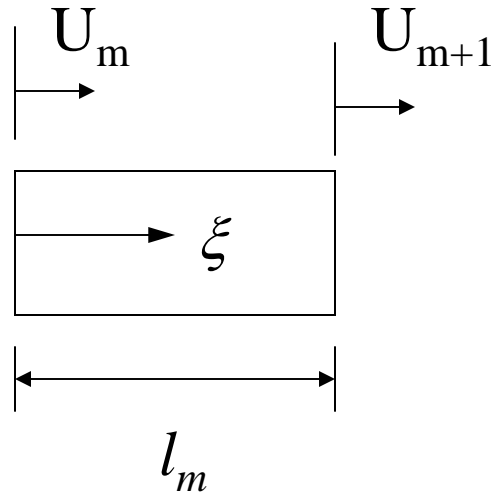
1-D Finite Elements – Virtual Work Approach



$q = \text{force/length} = \text{constant}$



l
mth element



1. Assume some variation of displacements in the element and write in terms of the nodal displacements

Simplest case: linear variation

$$\begin{aligned}
 u_x^{(m)} &= U_m + \frac{(U_{m+1} - U_m) \xi}{l_m} \\
 &= U_m \left(1 - \frac{\xi}{l_m} \right) + U_{m+1} \left(\frac{\xi}{l_m} \right)
 \end{aligned}$$

\uparrow
 $H_m^{(m)}(\xi)$

\uparrow
 $H_{m+1}^{(m)}(\xi)$

shape functions for displacement

If we let $H_j^{(m)} = 0 \quad j \neq m, m+1$ then we can write the displacement in each element in terms of all the nodal variables

$$u_x^{(m)} = \sum_{j=1}^{M+1} H_j^{(m)} U_j \quad (m = 1, \dots, M) \quad (\text{if we have } M \text{ elements we have } M+1 \text{ nodes})$$

2. Compute the strains for each element in terms of the nodal displacements

$$\begin{aligned} e_{xx}^{(m)} &= \frac{\partial u_x^{(m)}}{\partial x} = \frac{U_{m+1} - U_m}{l_m} \\ &= U_m \left(-\frac{1}{l_m} \right) + U_{m+1} \left(\frac{1}{l_m} \right) \\ &\quad \uparrow \qquad \qquad \uparrow \\ &\quad J_m^{(m)}(\xi) \quad J_{m+1}^{(m)}(\xi) \end{aligned}$$

shape functions for strain (constants in this case)

again, if we let $J_j^{(m)} = 0 \quad j \neq m, m+1$ then we can write the strain in each element in terms of all the nodal variables

$$e_{xx}^{(m)} = \sum_{j=1}^{M+1} J_j^{(m)} U_j \quad (m = 1, \dots, M)$$

3. Obtain virtual changes of the displacement and strain

$$\delta u_x^{(m)} = \sum_{j=1}^{M+1} H_j^{(m)} \delta U_j \quad (m = 1, \dots, M)$$

$$\delta e_{xx}^{(m)} = \sum_{j=1}^{M+1} J_j^{(m)} \delta U_j \quad (m = 1, \dots, M)$$

4. Apply Principle of Virtual Work

If $\int_V f_x \delta u_x dV + \sum P_k \delta U_k = \int_V \sigma_{xx} \delta e_{xx} dV$ is satisfied for all possible δU 's then equilibrium will be (approximately) satisfied

work done by: distributed body force concentrated loads

For concentrated loads, let $d_j = \begin{cases} 1 & \text{if load at node} \\ 0 & \text{otherwise} \end{cases}$ Then $\sum P_k \delta U_k = \sum_{j=1}^{M+1} P_j d_j \delta U_j$

and the principle gives

$$\sum_{m=1}^M \left\{ \int_{V_m} E \left[\sum_{k=1}^{M+1} J_k^{(m)} U_k \right] \left[\sum_{j=1}^{M+1} J_j^{(m)} \delta U_j \right] dV_m \right\} = \sum_{m=1}^M \left\{ \int_{V_m} f_x \left[\sum_{j=1}^{M+1} H_j^{(m)} \delta U_j \right] dV_m \right\} + \sum_{j=1}^{M+1} P_j d_j \delta U_j$$

$$\sigma_{xx}^{(m)} = E e_{xx}^{(m)} \quad \delta e_{xx}^{(m)} \quad \delta u_x^{(m)}$$

$$\sum_{m=1}^M \left\{ \int_{V_m} E \left[\sum_{k=1}^{M+1} J_k^{(m)} U_k \right] \left[\sum_{j=1}^{M+1} J_j^{(m)} \delta U_j \right] dV_m \right\} = \sum_{m=1}^M \left\{ \int_{V_m} f_x \left[\sum_{j=1}^{M+1} H_j^{(m)} \delta U_j \right] dV_m \right\} + \sum_{j=1}^{M+1} P_j d_j \delta U_j$$

Rewrite as:

$$\sum_{j=1}^{M+1} \delta U_j \left\{ \sum_{k=1}^{M+1} \left[\sum_{m=1}^M \int_{V_m} E J_j^{(m)} J_k^{(m)} dV_m \right] U_k - \sum_{m=1}^M \left[\int_{V_m} f_x H_j^{(m)} dV_m \right] - P_j d_j \right\} = 0$$

For this to be true for all variations of δU_j

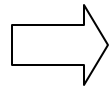
$$\sum_{k=1}^{M+1} \left[\sum_{m=1}^M \int_{V_m} E J_j^{(m)} J_k^{(m)} dV_m \right] U_k = \sum_{m=1}^M \left[\int_{V_m} f_x H_j^{(m)} dV_m \right] + P_j d_j \quad (j=1, 2, \dots, M+1)$$

$K_{jk}^{(m)}$
 stiffness matrix
 for each element

R_j
 nodal forces produced
 by distributed loads

$$K_{jk} = \sum_{m=1}^M K_{jk}^{(m)} \quad \text{total stiffness matrix}$$

$$\sum_{k=1}^{M+1} \left[\sum_{m=1}^M \int_{V_m} E J_j^{(m)} J_k^{(m)} dV_m \right] U_k = \sum_{m=1}^M \left[\int_{V_m} f_x H_j^{(m)} dV_m \right] + P_j d_j \quad (j=1, 2, \dots, M+1)$$



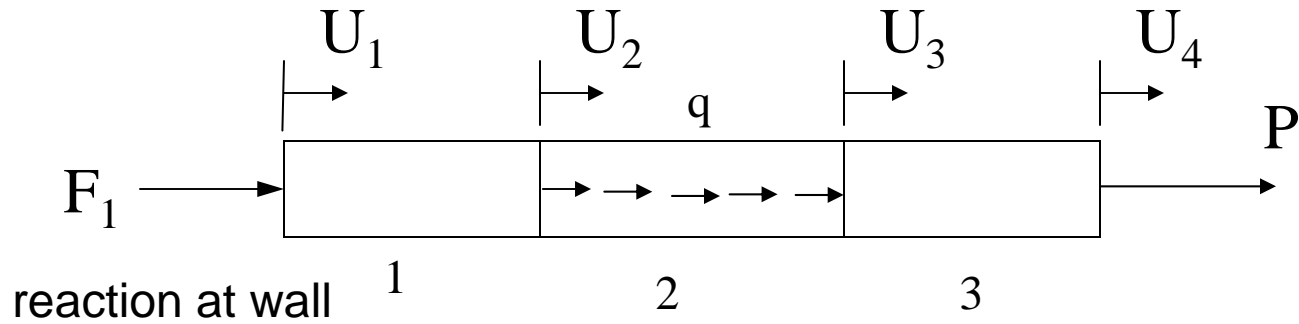
$$\sum_{k=1}^{M+1} K_{jk} U_k = R_j + P_j d_j$$

$$K_{jk} = \sum_{m=1}^M K_{jk}^{(m)}$$

$$R_j = \sum_{m=1}^M \left[\int_{V_m} f_x H_j^{(m)} dV_m \right]$$

$$K_{jk}^{(m)} = \sum_{m=1}^M \int_{V_m} E J_j^{(m)} J_k^{(m)} dV_m$$

Now, consider our particular problem



$$\begin{bmatrix} K_{11} & K_{12} & K_{13} & K_{14} \\ K_{21} & K_{22} & K_{23} & K_{24} \\ K_{31} & K_{32} & K_{33} & K_{34} \\ K_{41} & K_{42} & K_{43} & K_{44} \end{bmatrix} \begin{Bmatrix} U_1 \\ U_2 \\ U_3 \\ U_4 \end{Bmatrix} = \begin{Bmatrix} R_1 \\ R_2 \\ R_3 \\ R_4 \end{Bmatrix} + \begin{Bmatrix} F_1 \\ 0 \\ 0 \\ P \end{Bmatrix}$$

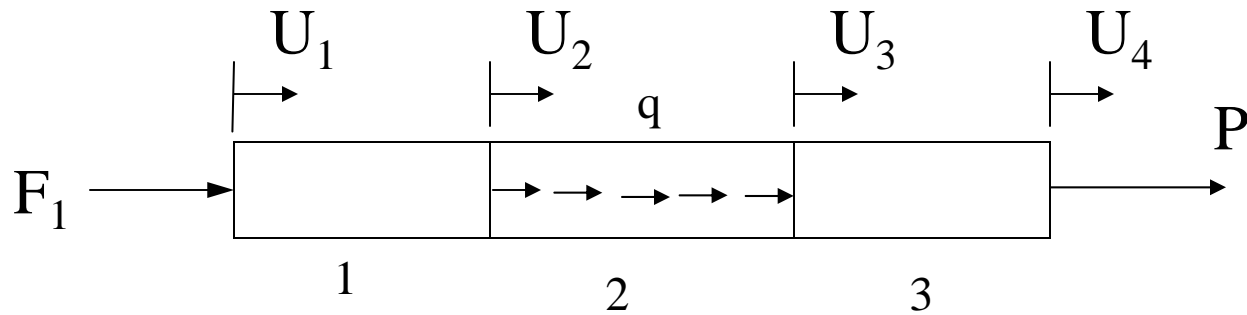
6. Obtain stiffness matrix for each element:

$$\begin{aligned}
 K_{jk}^{(m)} &= \sum_{m=1}^M \int_{V_m} E J_j^{(m)} J_k^{(m)} dV_m & J_m^{(m)}(\xi) &= -\frac{1}{l} \\
 &= \sum_{m=1}^3 EA l J_j^{(m)} J_k^{(m)} & J_{m+1}^{(m)}(\xi) &= \frac{1}{l} \\
 & & \text{all other } J_j^{(m)}(\xi) &= 0
 \end{aligned}$$

$$K_{jk}^{(1)} : \begin{bmatrix} \frac{EA}{l} & -\frac{EA}{l} & 0 & 0 \\ -\frac{EA}{l} & \frac{EA}{l} & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \quad K_{jk}^{(2)} : \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & \frac{EA}{l} & -\frac{EA}{l} & 0 \\ 0 & -\frac{EA}{l} & \frac{EA}{l} & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \quad K_{jk}^{(3)} : \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & \frac{EA}{l} & -\frac{EA}{l} \\ 0 & 0 & -\frac{EA}{l} & \frac{EA}{l} \end{bmatrix}$$

7. Assemble total stiffness matrix:

$$K_{jk} = \sum_{m=1}^3 K_{jk}^{(m)} : \begin{bmatrix} \frac{EA}{l} & -\frac{EA}{l} & 0 & 0 \\ -\frac{EA}{l} & \frac{2EA}{l} & -\frac{EA}{l} & 0 \\ 0 & -\frac{EA}{l} & \frac{2EA}{l} & -\frac{EA}{l} \\ 0 & 0 & -\frac{EA}{l} & \frac{EA}{l} \end{bmatrix}$$



7. Obtain all equivalent nodal forces from distributed loads:

$$R_j = \sum_{m=1}^M \left[\int_{V_m} f_x H_j^{(m)} dV_m \right]$$

$$= \int_{V_2} f_x H_j^{(2)} A d\xi$$

$$= q \int_0^l H_j^{(2)}(\xi) d\xi$$

$$H_2^{(2)}(\xi) = 1 - \frac{\xi}{l}$$

$$H_3^{(2)}(\xi) = \frac{\xi}{l}$$

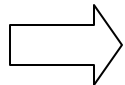
$$\text{all other } H_j^{(2)}(\xi) = 0$$

$$R_1 = 0$$

$$R_2 = q \int_0^l \left(1 - \frac{\xi}{l} \right) d\xi = \frac{ql}{2}$$

$$R_3 = q \int_0^l \left(\frac{\xi}{l} \right) d\xi = \frac{ql}{2}$$

$$R_4 = 0$$



Resulting system cannot be solved directly because we have not eliminated rigid body motions

$$\begin{bmatrix} \frac{EA}{l} & \frac{-EA}{l} & 0 & 0 \\ \frac{-EA}{l} & \frac{2EA}{l} & \frac{-EA}{l} & 0 \\ 0 & \frac{-EA}{l} & \frac{2EA}{l} & \frac{-EA}{l} \\ 0 & 0 & \frac{-EA}{l} & \frac{EA}{l} \end{bmatrix} \begin{Bmatrix} U_1 \\ U_2 \\ U_3 \\ U_4 \end{Bmatrix} = \begin{Bmatrix} 0 \\ ql/2 \\ ql/2 \\ 0 \end{Bmatrix} + \begin{Bmatrix} F_1 \\ 0 \\ 0 \\ P \end{Bmatrix}$$

8. Eliminate all possible rigid body motions and solve the system of equations

For this problem, if $U_j = 0$ set $K_{jj} = 1$, $K_{jk} = 0$ ($k \neq j$) and $R_j = P_j d_j = 0$

so that the j th equation gives $U_j = 0$ To preserve symmetry of the

stiffness matrix, also set $K_{kj} = 0$

$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & \frac{2EA}{l} & \frac{-EA}{l} & 0 \\ 0 & \frac{-EA}{l} & \frac{2EA}{l} & \frac{-EA}{l} \\ 0 & 0 & \frac{-EA}{l} & \frac{EA}{l} \end{bmatrix} \begin{Bmatrix} U_1 \\ U_2 \\ U_3 \\ U_4 \end{Bmatrix} = \begin{Bmatrix} 0 \\ ql/2 \\ ql/2 \\ 0 \end{Bmatrix} + \begin{Bmatrix} 0 \\ 0 \\ 0 \\ P \end{Bmatrix}$$