

BLADE ELEMENT THEORY

Assumptions:

- The blade is composed of aerodynamically independent, narrow strips or elements.
- A differential blade element of chord C and width dr , located at a radius r from the rotor axis is considered as an airfoil section.

Theory:

The airscrew is advancing at a speed of V and the velocity at the disc is given by $V_d = V_0(1 + a) = V_0 + v$.

The slipstream behind an airscrew rotates in the same sense as the blades about the airscrew axis (z-axis).

The angular velocity of the airscrew blades are Ω and the angular velocity of the flow in the plane of the blades is $a_1\Omega$ where a_1 is a constant for the element considered.

This element has a linear velocity in the plane of the rotation of Ωr and the flow is itself rotating in the same plane and sense with an angular velocity $a_1\Omega$.

Thus the relative linear velocity of the element relative to the air in this plane is $\Omega r(1 - a_1)$ or $r(\Omega - \omega)$.

θ is the geometric helix angle of the element measured between the zero lift-line of the element and the rotor disc.

α is the angle between the relative velocity V_R and the chord.

ϕ is the angle between the resultant velocity V_R and the plane of rotation.

Then from the geometry one can deduce:

$$\theta = \phi - \alpha \quad \phi = \arctan \frac{V_d}{\Omega r - \omega} = \arctan \frac{V_0(1+a)}{\Omega r(1-a_1)}$$

or

$$\alpha = \theta - \arctan \frac{V_0 + v}{\Omega r - \omega} \quad (1)$$

STATIONARY BLADE ELEMENT WITH AIR FLOWING PAST IT

The elemental lift expressed by the blade element is:

$$\partial L = \frac{1}{2} \rho V_R^2 C_l c dr \quad (2)$$

Where $V_R^2 = (V + \omega)^2 + (r * (\Omega - \omega))^2$

The elemental drag is found to be:

$$\partial D = \frac{1}{2} \rho V_R^2 C_d c dr \quad (3)$$

Where C_l and C_d are 2-D aerodynamic characteristics of the blade section. From the force diagram

$$\partial T = \partial L \cos \phi - \partial D \sin \phi \quad (4)$$

If B is the number of blades the $dT = b\partial T$.

i.e

$$dT = bc \frac{1}{2} \rho V_R^2 (C_l \cos \phi - C_d \sin \phi) dr \quad (5)$$

Also from the force diagram: $\frac{\partial Q}{r} = \partial L \sin \phi + \partial D \cos \phi$

where ∂Q is the torque required to rotate the element about the axis of rotation.

Following the procedure used for dT one can show that the torque required to rotate B blades is given by:

$$dQ = Bc \left(\frac{1}{2} \rho V_R^2\right) (C_l \sin \phi + C_d \cos \phi) r dr \quad (6)$$

From dT and dQ one can obtain the thrust (T), torque (Q), and power required (P) using the equations given below:

$$T = \int_0^R dT \quad (7)$$

$$Q = \int_0^R dQ \quad (8)$$

$$P = Q\Omega \quad (9)$$

$\frac{dT}{dr}$ and $\frac{dQ}{dr}$ are known as the thrust grading and the torque grading respectively.

Thrust dT from momentum principle: $dT = \dot{m}\delta V = (\text{area of annulus} * \text{velocity} * \text{density}) \delta v$

or

$$\begin{aligned} dT &= (2\pi r dr * V(1+a)\rho)(V_e - V_0) \quad dT = (2\pi r dr * V(1+a)\rho)(v_0(1+2a) - V_0) \\ &= (2\pi r \rho V_0^2(1+a)(2a)dr \end{aligned} \quad (10)$$

Equating 5 and 10

$$bc\left(\frac{1}{2}\rho V_R^2\right)(C_i \cos \phi - C_d \sin \phi)dr = 2\pi r \rho V_0^2(1+a)(2a)dr$$

upon rearrangement:

$$\frac{bc}{4\pi r} V_R^2 (C_l \cos \phi - C_d \sin \phi) = V_0^2 (1 + a)(2a) \quad (11)$$

but from the velocity diagram:

$$V_R = V_0(1 + a)1/\sin \phi \quad (12)$$

substitute (12) in (11) and rearrange to get:

$$\frac{bc}{8\pi r} 1/\sin^2 \phi (C_l \cos \phi - C_d \sin \phi) = \frac{a}{1 + a} \quad (13)$$

Relative angular velocity of the flow far upstream is Ω

Relative angular velocity of the flow at the disc is $\Omega - a_1\Omega$ or $\Omega - \omega$

Relative angular velocity of the flow far down stream is $\Omega - 2a_1\Omega$ or $\Omega - 2\omega$.

Elemental torque in the annulus dQ is equal to the angular momentum change per unit time in the annulus = $\dot{m}(\delta V_T)r$

$$dQ = (2\pi r dr \rho V_0(1 + a)(2a_1\Omega r)(r)$$

$$dQ = 4\pi r^3 \rho V_0(1 + a)\Omega a_1 dr \quad (14)$$

Equating 6 and 14

$$bc r \frac{1}{2} \rho V_R^2 (C_l \sin \phi + C_d \cos \phi) dr$$

$$= 4\pi r^3 \rho v_0(1 + a)\Omega a_1 dr \quad (15)$$

From the velocity diagram:

$$V_R = \Omega r(1 - a_1) \sec \phi \quad (16)$$

Substitute 12 and 16 into equation 15 and rearrange to get

$$\frac{bc}{8\pi r} (1/\sin \phi)(1/\cos \phi)(C_l \sin \phi + C_d \cos \phi) = \frac{a_1}{1 - a_1} \quad (17)$$

Summary

- Assume a and a_1
- Calculate $V_0(1 + a)$ and $\Omega r(1 - a_1)$ the velocity in the axial and tangential edirection at the plane of rotation.
- Calculate α and ϕ where $\phi = \arctan \frac{V_0 + v}{(\Omega - \omega)r}$ $\alpha = \theta - \phi$
- calculate C_l and C_d from 2-D sectional characteristics of the airfoil used.
- calculate new a and a_1 using : $\frac{a}{a+1} = \frac{bc}{8\pi r} 1/\sin^2 \phi (C_l \cos \phi - C_d \sin \phi)$
 $\frac{a_1}{1-a_1} = \frac{bc}{8\pi r} 1/\cos \phi \sec \phi (C_l \sin \phi + C_d \cos \phi)$
- if a and a_1 are converged go to step seven otherwise go to step 2 with new values of a and a_1 . (For numerical convergence take an arithmetic mean of old a and a_1)
- calculate T , Q , P and C_T , C_Q , and C_P